

FRACTURE PROPAGATION IN THE INTERNAL PRESSURIZED VESSEL

APPROVED

By CO HONG TRAN at 1:18 pm, Feb 25, 2010



by CO. H. TRAN .

MMPC HCMC
Vietnam

coth123@math.com & cohtran@math.com

Copyright 2009

November 06 2009

**** Abstract :**

- Calculating the maximum value P_{max} that causes the fracture propagation while applying the internal static pressure .
- Evaluating the time-limit for applying the continuous / discrete form of internal dynamic pressure for the round metal vessel under conditions given by series expansion and curve -fitting method .

**** Subjects:** Fracture Mechanics , The stress intensity factor K_I .

Introduction

This worksheet demonstrates Maple's capabilities in estimating the maximum pressure and the time-limit for applying static or dynamic pressure on the surface of a round metal vessel which has an internal semi-elliptical surface crack .

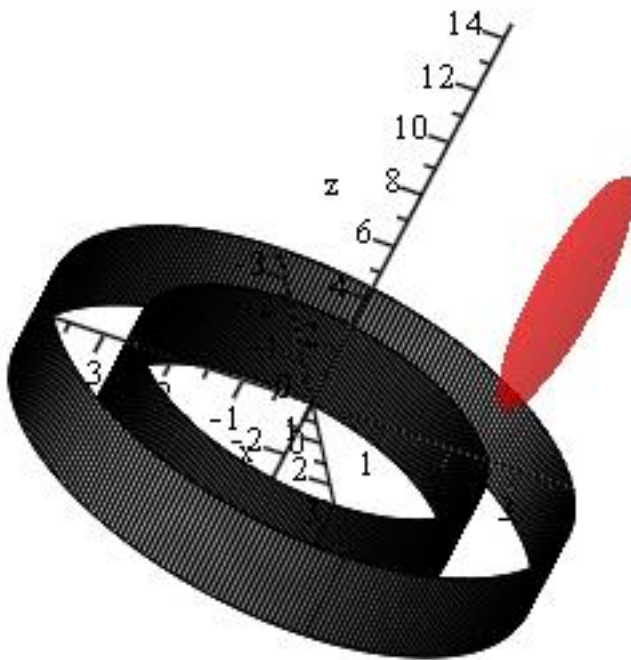
All rights reserved. Copying or transmitting of this material without the permission of the authors is not allowed .

1. Modeling the problem

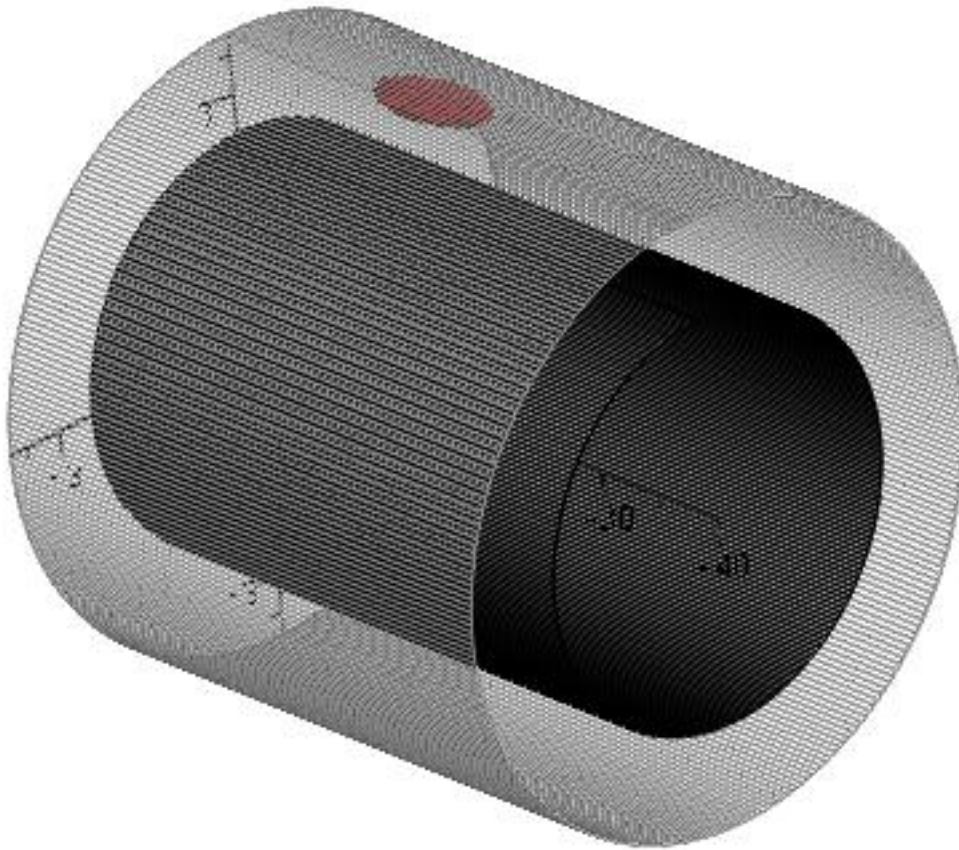
We consider the round metal vessel with parameters given :

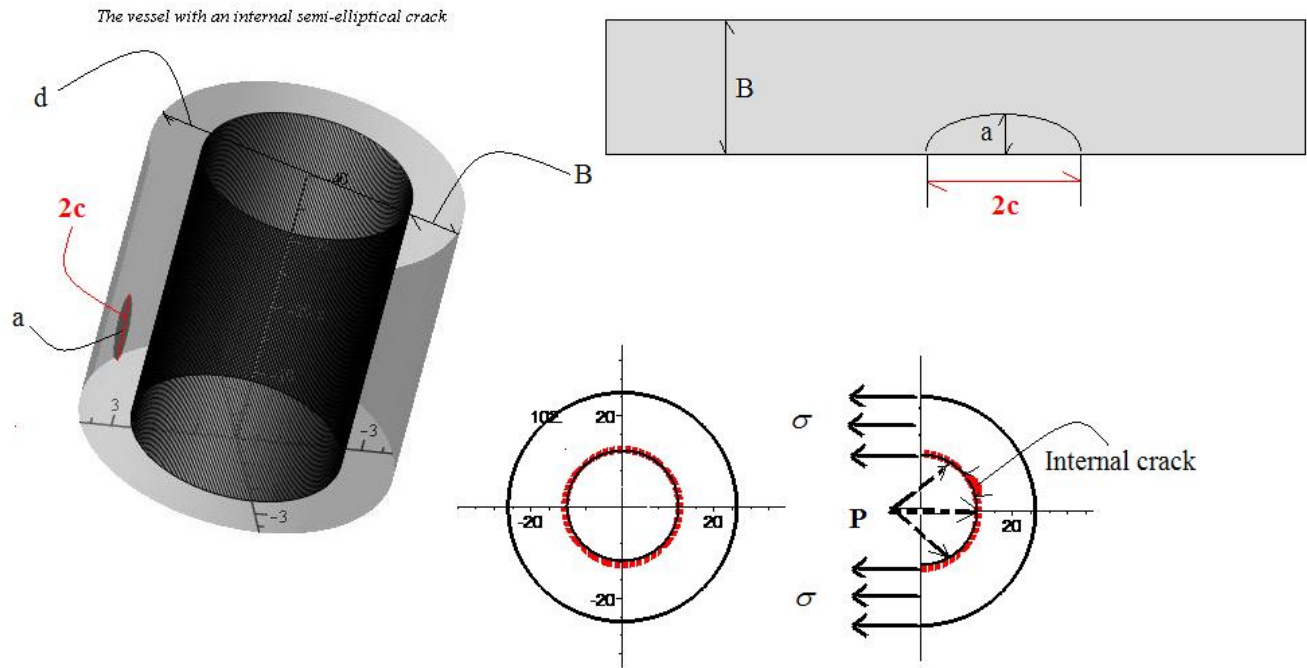
a : one-half crack depth (mm) ; B : thickness of vessel (mm) ; c : one-half crack length (mm) ; d : internal diameter (mm) ; P : internal pressure (MPa) .

*Modeling the sizes of the vessel **and** crack*



The vessel with an internal semi-elliptical crack





[>
=>

2. Technical basis and calculation formulas

Depending on the principle of superposition , the total stress intensity factor will be determined by :

$$K_{I total} = K_{I Hoop} + K_{I Pressure}$$

After calculating , $K_{I total}$ can be rewritten in the form : $K_{I total} := \alpha \cdot P_{pressure} \cdot \sqrt{a\pi}$

$$\left(\alpha : correction factor = \frac{1.12}{\sqrt{Q_{shape}}} \left(\frac{d_{diameter}}{2 \cdot B_{thickness}} + 1 \right) \right)$$

Use the formulas of hoop stress : $\sigma_{hoop} = \frac{1}{2} \frac{P_{press} d_{diameter}}{B_{thickness}}$ and

the shape factor $Q_{shape} = \frac{1}{4} \pi^2 \left(\frac{3}{4} + \frac{1}{4} \frac{a^2}{c^2} \right)^2 - \frac{7}{33} \frac{\sigma^2}{\sigma_{ys}^2}$, we find the value of correction factor

α , consequently it gives the total stress intensity $K_{I total}$. Because the critical crack length is

defined by : $a_{crack} = \frac{1}{\pi} \cdot \left(\frac{K_{Ic}}{\alpha_{correction} \cdot \sigma} \right)^2$, we may conclude that the maximum crack length has a

interactive relation with the stress intensity factor $K_{I total}$.

Following is the table which shows temperature mechanical properties for some materials used in engineering design .

Table 1.

<i>Material</i>	σ_{ys}	σ_{ts}	K_{Ic}	E
<i>AerMet 100</i>	1724	1965	126	207
<i>Ti – 6 Al – 4 V 1</i>	869	958	87	117
<i>Ti – 6 Al – 4 V 2</i>	1007	1034	40	130
<i>AISI 4340 1</i>	1089	1097	110	207
<i>AISI 4340 2</i>	1476	1896	81	207
<i>AISI 4147</i>	945	1062	120	207
<i>INCONEL</i>	1172	1404	96	–
<i>18 Ni (250)</i>	1290	1345	176	–
<i>2014 – T651</i>	455	–	24	–
<i>2024 – T3</i>	345	–	44	–
<i>7075 – T6</i>	572	641	24	72
<i>Ni₄₉ Fe₂₉ P₁₄ B₆ Si₂</i>	800	800	12	–
<i>SIC</i>	460	460	3.7	72

3. Construct algorithms for applications

Here the author will present two procedures that are used to calculate the maximum pressure (P_{max}) causing the fracture propagation and estimate the limitation of time while applying the continuous or discrete internal pressure functions for the round metal vessel under the given constraints .

3.1 Calculating the maximum pressure :

The matrix A below is used to define the mechanical properties of each type of metal which is required for engineering design .

$pressure_{Max} := \text{proc}(a1, B1, c1, d1, P1, type1, k)$

the data of problem will be entered into the procedure with the instructions below

a1 : one-half crack depth (mm) ,

B1 : thickness of vessel (cm) ,

c1 : one-half crack length (mm) ,

d1 : internal diameter(m) ,

P1 : internal pressure (MPa) ,

type1 : type of material (inserted in the form of an element of the matrix A i.e A [k][i]) ,

k : the kth row (see table 1.) .

```
> restart, A := Matrix([ [ material,  $\sigma_{ys}$ ,  $\sigma_{ts}$ ,  $K_{IC}$ ,  $E$  ], [AerMet100, 1724, 1965, 126, 207 ], [Ti6Al4V1, 869, 958, 87, 117 ], [Ti6Al4V2, 1007, 1034, 40, 130 ], [AISI4340I, 1089, 1097, 110, 207 ], [AISI43402, 1476, 1896, 81, 207 ], [AISI4147, 945, 1062, 120, 207 ], [INCONEL, 1172, 1404, 96, NA ], [18 Ni250, 1290, 1345, 176, NA ], [2014 T65I, 455, NA, 24, NA ], [2024 T3, 345, NA, 44, NA ], [7075 T6, 572, 641, 24, 72 ], [Ni49Fe29P14B6Si2, 800, 800, 12, NA ], [SIC, 460, 460, 3.7, 72 ]])
```

$A := \left[\begin{array}{l} 14 \times 5 \text{ Matrix} \\ \text{Data Type: anything} \\ \text{Storage: rectangular} \\ \text{Order: Fortran_order} \end{array} \right]$

(1)

```
> pressure_Max := proc(a1, B1, c1, d1, P1, type1, k) local i, s, anew, cnew, Bnew, sigma,
sigma1, alpha1, alphax, P, Q, Q1, K, KI1, KIC; with(plots, implicitplot) :
print(" Start the program "); print(" The parameters are given : ", a = a1, B = B1, 2 c = 2
· c1, d = d1, P = P1, type = type1,  $\frac{a}{2 \cdot c} = \text{evalf}\left(\frac{a1}{2 \cdot c1}, 3\right)$ , row = k);
print(" Convert the units into meter : "); anew :=  $\frac{a1}{1000}$ ; Bnew :=  $\frac{B1}{100}$ ; cnew
:=  $\frac{c1}{1000}$ ; print(a = anew, B = Bnew, c = cnew, d = d1);  $K_I := \text{alpha} \cdot P \cdot \sqrt{\pi \cdot a}$ ;
print(" (1) The total stress intensity factor KI " =  $\text{alpha} \cdot P_{\text{pressure}} \cdot \sqrt{\pi \cdot a}$ ); ; alphax
:=  $\frac{1.12}{\sqrt{Q_{\text{shape}}}} \cdot \left( \frac{d_{\text{diameter}}}{2 \cdot B_{\text{thickness}}} + 1 \right)$ ; print(" (2) Correction factor  $\alpha$  " =  $\frac{1.12}{\sqrt{Q_{\text{shape}}}} \cdot \left( \frac{d}{2 \cdot B} + 1 \right)$ );  $Q_{\text{shape}} := \frac{\pi^2}{4} \cdot \left( \frac{3}{4} + \left( \frac{a}{2 \cdot c} \right)^2 \right)^2 - \frac{7}{33} \cdot \left( \frac{\sigma}{\sigma_{ys}} \right)^2$ ; sigma :=  $\frac{P_{\text{pressure}} \cdot d_{\text{diameter}}}{2 \cdot B_{\text{thickness}}}$ ;
print(" (3) The hoop stress  $\sigma$  " =  $\frac{P_{\text{pressure}} \cdot d_{\text{diameter}}}{2 \cdot B_{\text{thickness}}}$ ); ;
print(" (4) The shape factor Qshape " =  $Q_{\text{shape}}$ ); printf("\n%s",
" Read the data from the temperature mechanical properties table"); printf("\n%s", " ");;
print(TYPE); for i from 1 to 5 do print(s[i] = A[k][i]); s[i] := A[k][i]; od; ;
print( Material = s[1],  $\Sigma_{ys} = s[2]$ ,  $\Sigma_{ts} = s[3]$ ,  $KIc = s[4]$ ,  $E = s[5]$ ); ; ; printf("\n%s",
" Calculating the hoop stress sigma , the shape factor Q , correction factor alpha and the
total stress intensity factor KI "); printf("\n%s", " ");; sigma1 :=  $\text{evalf}\left(\frac{P1 \cdot d1}{2 \cdot Bnew}, 5\right)$ ;
print(" From (3) "); print(" (3) The hoop stress  $\sigma$  " =  $\frac{P_{\text{pressure}} \cdot d_{\text{diameter}}}{2 \cdot B_{\text{thickness}}}$ );
print("  $\sigma$  " = sigma1); ; printf("\n%s", " ");; print(" From (4) ");;
print(" (4) The shape factor Q " =  $Q_{\text{shape}}$ );  $\sigma_{ys} := s[2]$ ;  $Q1 := \frac{\pi^2}{4} \cdot \left( \frac{3}{4} \right)$ 
```

```

+ \left( \frac{a_{new}}{2 \cdot c_{new}} \right)^2 \Big)^2 - \frac{7}{33} \cdot \left( \frac{\sigma l}{\sigma_{ys}} \right)^2 ; QI := evalf(QI, 3); ; print(" Qshape " = QI); ;

printf("\n%s", " "); print(" From (2) "); print\left( "(2) Correction factor \alpha " = \frac{1.12}{\sqrt{Q_{shape}}} \cdot \left( \frac{d_{diameter}}{2 \cdot B_{thickness}} + 1 \right) \right); alphaI := \frac{1.12}{\sqrt{QI}} \cdot \left( \frac{dl}{2 \cdot B_{new}} + 1 \right); alphaI := evalf(alphaI, 3);

print(" \alpha " = alphaI); ; printf("\n%s", " "); print(" From (1) ");

print\left( " (1) The total stress intensity KI " = \alpha \cdot P_{pressure} \cdot \sqrt{\pi \cdot a} \right); KII := alphaI \cdot P1 \cdot \sqrt{\pi \cdot a_{new}}; print(KI = KII); KII := evalf(KII, 4); print(KI = KII); ; printf("\n%s", " "); printf("\n%s", " 1/. Compare the total stress intensity factor KI and KIc "); ;

printf("\n%s", " "); KIC := s[4]; print(KIc = KIC); if KII < KIC
then print(" The fracture will not occur / the crack propagation is stable , because KI < KIc "); ; printf("\n%s", " "); ; printf("\n%s", " "); printf("\n%s", " 2/. Determine the maximum pressure Pmax that causes the crack propagation "); ;

printf("\n%s", " "); print(" From (1) "); print\left( " (1) The critical stress intensity KIc " = \alpha[c] \cdot P_{Max} \cdot \sqrt{\pi \cdot a} \right); P_{MAX} := \frac{KIC}{\alpha I \cdot \sqrt{\pi \cdot a_{new}}}; P_{MAX} := evalf(P_{MAX}, 5); print(Pmax = P_{MAX}); ; printf("\n%s", " "); printf("\n%s", " ");

" 3/. Consider the relation between Q and \alpha "; ; printf("\n%s", " ");
print(" From (2) "); print(" Substitute x for alpha and y for Qshape into (2) ");

print(alpha = alphax); x = evalf\left( \frac{1.12}{\sqrt{y}} \cdot \left( \frac{dl}{2 \cdot B_{new}} + 1 \right), 4 \right); print(%o, plot1) ; y = x \cdot P1 \cdot \sqrt{\pi \cdot a_{new}}; ; print(%o, plot2); ; printf("\n%s", " "); ; printf("\n%s", " ");

print(" Graphical relations between correction and shape factor (plot1) _ correction and stress intensity factor KI (plot2) " ); ; print(" End of program " ); ; printf("\n%s", " ");

plot\left( \left[ evalf\left( \frac{1.12}{\sqrt{x}} \cdot \left( \frac{dl}{2 \cdot B_{new}} + 1 \right), 4 \right), x \cdot P1 \cdot \sqrt{\pi \cdot a_{new}} \right], x = 0 ..4, y = 0 ..20, axes = boxed, labels = [ `Q`, `KI`, `alpha` ], color = [blue, red], legend = [plot1, plot2], thickness = [2, 1] \right);

else print(" Warning ! The fracture will occur / the crack propagation is unstable , because KI >= KIc " ); ; print(" End of program " ); ; fi;
end:

```

Example 3.1.1

Consider a round vessel with technical data below :

one-half crack depth $a = 5.5\text{mm}$; thickness of vessel $B = 20\text{cm}$; one-half crack length $c = 4.5\text{mm}$; internal diameter $d = 0.5\text{m}$; internal pressure $P = 362\text{ MPa}$; type : AISI 4340 2 i.e A[6][1]

Calculate :

a. The stress intensity factor KI .

b. The maximum static pressure that would cause the fracture propagation .
 Compare the total stress intensity factor KI and K_{Ic} (see table 1.) and give your conclusion .

Solution :

Run the procedure *

```

> pressure_Max(5.5, 20, 4.5, 0.5, 362, A[6][1], 6);
                                " Start the program "
" The parameters are given : ", a = 5.5, B = 20, 2 c = 9.0, d = 0.5, P = 362, type = AISI43402,
  1/2 * a/c = 0.610, row = 6
                                " Convert the units into meter : "
a = 0.005500000000, B = 1/5, c = 0.004500000000, d = 0.5
" (1) The total stress intensity factor KI " = alpha * P_pressure * sqrt(pi * a)
                                " (2) Correction factor alpha " = 1.12 * (1/2 * d/B + 1) / sqrt(Q_shape)
                                " (3) The hoop stress sigma " = 1/2 * P_pressure * d_diameter / B_thickness
                                " (4) The shape factor Q_shape " = 1/4 * pi^2 * (3/4 + 1/4 * d^2/c^2)^2 - 7/33 * sigma_y^2/sigma_y^2

Read the data from the temperature mechanical properties table

                                TYPE
s1 = AISI43402
s2 = 1476
s3 = 1896
s4 = 81
s5 = 207

Material = AISI43402, Sigma_ys = 1476, Sigma_ts = 1896, K_Ic = 81, E = 207

Calculating the hoop stress sigma , the shape factor Q ,
correction factor alpha and the total stress intensity factor
KI

                                " From (3) "

```


$$\text{" (3) The hoop stress } \sigma \text{"} = \frac{1}{2} \frac{P_{pressure} d_{diameter}}{B_{thickness}}$$

$$\text{" } \sigma \text{"} = 452.50$$

" From (4) "

$$\text{" (4) The shape factor } Q \text{"} = \frac{1}{4} \pi^2 \left(\frac{3}{4} + \frac{1}{4} \frac{a^2}{c^2} \right)^2 - \frac{7}{33} \frac{\sigma^2}{\sigma_{ys}^2}$$

$$\text{" } Q_{shape} \text{"} = 3.10$$

" From (2) "

$$\text{"(2) Correction factor } \alpha \text{"} = \frac{73.92 \left(\frac{1}{2} \frac{d_{diameter}}{B_{thickness}} + 1 \right)}{\sqrt{1089 \pi^2 \left(\frac{3}{4} + \frac{1}{4} \frac{a^2}{c^2} \right)^2 - \frac{924 \sigma^2}{\sigma_{ys}^2}}}$$

$$\text{" } \alpha \text{"} = 1.43$$

" From (1) "

$$\text{" (1) The total stress intensity } KI \text{"} = \alpha P_{pressure} \sqrt{\pi a}$$

$$KI = 38.39069309 \sqrt{\pi}$$

$$KI = 68.07$$

1/. Compare the total stress intensity factor KI and KIc

$$KIc = 81$$

" The fracture will not occur / the crack propagation is stable , because $KI < KIc$ "

2/. Determine the maximum pressure Pmax that causes the crack propagation

" From (1) "

$$\text{" (1) The critical stress intensity } KIc \text{"} = \alpha_c P_{Max} \sqrt{\pi a}$$

$$P_{max} = 430.90$$

3/. Consider the relation between Q and α ;

" From (2) "

" Substitute x for α and y for Qshape into (2) "

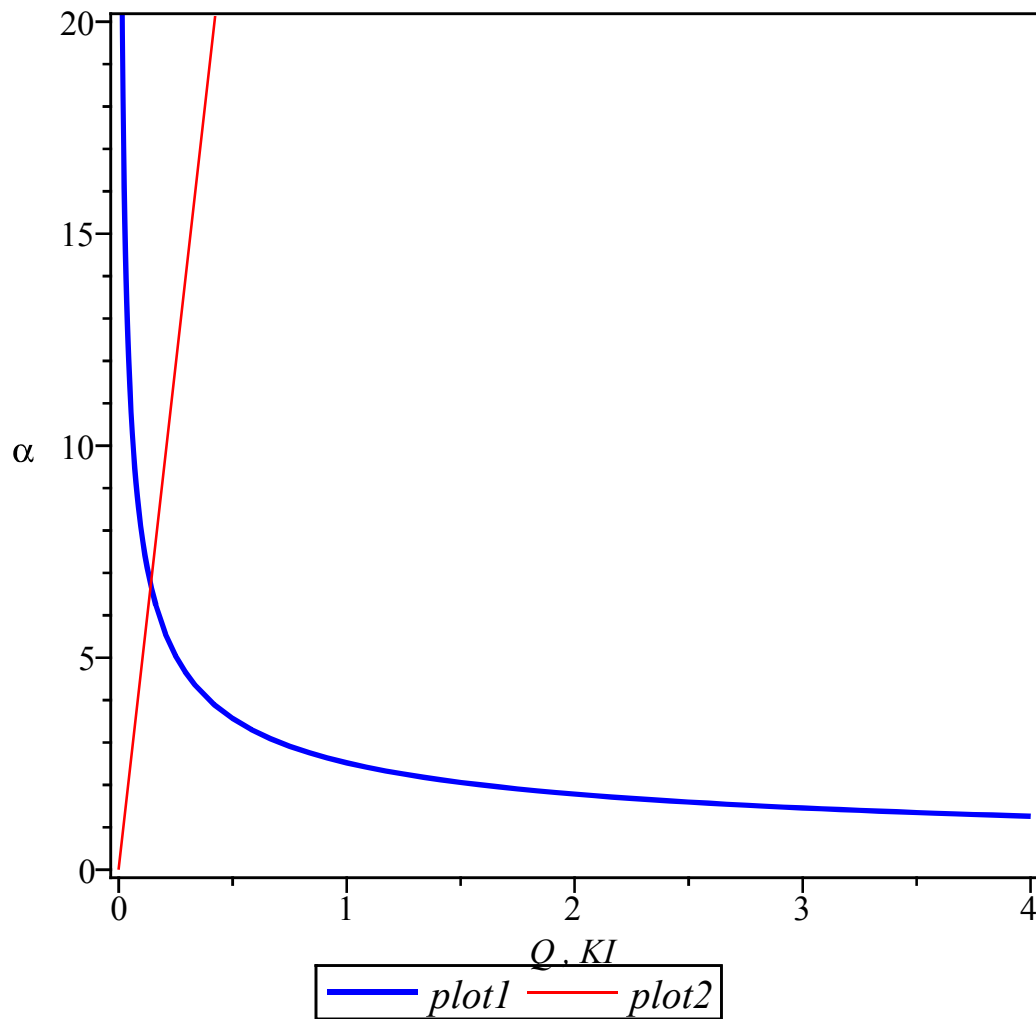
$$\alpha = \frac{1.12 \left(\frac{1}{2} \frac{d_{diameter}}{B_{thickness}} + 1 \right)}{\sqrt{Q_{shape}}}$$

$$x = \frac{2.520}{\sqrt{y}}, plot1$$

$$y = 26.84663852 x \sqrt{\pi}, plot2$$

" Graphical relations between correction and shape factor (plot1) _ correction and stress intensity factor KI (plot2) "

" End of program "



Example 3.1.2

Consider a round vessel with technical data below :

one-half crack depth $a = 4.5\text{mm}$; thickness of vessel $B = 20\text{cm}$; one-half crack length $c = 6\text{mm}$; internal diameter $d = 0.8\text{m}$; internal pressure $P = 560\text{ MPa}$; type : AISI 4340 1 i.e $A[5][1]$

Calculate :

a. The stress intensity factor KI .

b. The maximum static pressure that would cause the fracture propagation .

Compare the total stress intensity factor KI and KIc (see table 1.) and give your conclusion .

Solution :

Run the procedure $pressure_{Max}(, , ,)$;

```
> pressure_Max(4.5, 20, 6, 0.8, 560, A[5][1], 5);
```

" Start the program "

" The parameters are given : ", $a = 4.5$, $B = 20$, $2c = 12$, $d = 0.8$, $P = 560$, $type = AISI43401$,

$$\frac{1}{2} \frac{a}{c} = 0.375, row = 5$$

" Convert the units into meter : "

$$a = 0.004500000000, B = \frac{1}{5}, c = \frac{3}{500}, d = 0.8$$

" (1) The total stress intensity factor KI " = $\alpha P_{pressure} \sqrt{\pi a}$

"(2) Correction factor α " = $\frac{1.12 \left(\frac{1}{2} \frac{d}{B} + 1 \right)}{\sqrt{Q_{shape}}}$

" (3) The hoop stress σ " = $\frac{1}{2} \frac{P_{pressure} d_{diameter}}{B_{thickness}}$

" (4) The shape factor Q_{shape} " = $\frac{1}{4} \pi^2 \left(\frac{3}{4} + \frac{1}{4} \frac{a^2}{c^2} \right)^2 - \frac{7}{33} \frac{\sigma_{ys}^2}{\sigma_{ys}^2}$

Read the data from the temperature mechanical properties table

TYPE

$$s_1 = AISI4340I$$

$$s_2 = 1089$$

$$s_3 = 1097$$

$$s_4 = 110$$

$$s_5 = 207$$

$$Material = AISI4340I, \Sigma_{ys} = 1089, \Sigma_{ts} = 1097, K_{Ic} = 110, E = 207$$

Calculating the hoop stress σ , the shape factor Q ,
correction factor α and the total stress intensity factor
KI

" From (3) "

" (3) The hoop stress σ " = $\frac{1}{2} \frac{P_{pressure} d_{diameter}}{B_{thickness}}$

$$\sigma = 1120.0$$

" From (4) "

" (4) The shape factor Q " = $\frac{1}{4} \pi^2 \left(\frac{3}{4} + \frac{1}{4} \frac{a^2}{c^2} \right)^2 - \frac{7}{33} \frac{\sigma_{ys}^2}{\sigma_{ys}^2}$

$$Q_{shape} = 1.73$$

" From (2) "

$$\alpha = \frac{73.92 \left(\frac{1}{2} \frac{d_{diameter}}{B_{thickness}} + 1 \right)}{\sqrt{1089 \pi^2 \left(\frac{3}{4} + \frac{1}{4} \frac{a^2}{c^2} \right)^2 - \frac{924 \sigma_{ys}^2}{\sigma_{ys}^2}}}$$

" α " = 2.55

" From (1) "

" (1) The total stress intensity KI " = $\alpha P_{pressure} \sqrt{\pi a}$

$$KI = 95.79315215 \sqrt{\pi}$$

$$KI = 169.8$$

1/. Compare the total stress intensity factor KI and KIc

$$KIc = 110$$

" Warning ! The fracture will occur / the crack propagation is unstable , because $KI \geq KIc$ "

" End of program "

(2)

3.2 Estimating the interval of time while applying the internal dynamic pressure

For the next procedure $pressure_{Estm} := \text{proc}(a1, B1, c1, d1, P1, type1, k, tlim, t0, y0, y1)$

we enter the data of the problem with following instructions

a1 : one-half crack depth (mm) ,
 B1 : thickness of vessel cm) ,
 c1 : one-half crack length (mm) ,
 d1 : internal diameter(m) ,
 P1 : internal pressure function (continous /discrete) (MPa) ,
 type1 : type of material (inserted in the form of an element of the matrix A i.e A [k][i]) ,
 k : the kth row (see table 1.) ,
 tlim : the upper bound of interval (0 , tlim) that is a constraint of time used to solve the inequality $KI < KIc$,
 t0, y0 , y1 : the minimum and maximum values of t and y used to plot the graph of funtions KI and the polynomial KITaylor .

> $pressure_{Estm} := \text{proc}(a1, B1, c1, d1, P1, type1, k, tlim, t0, y0, y1)$ **global** P2 , KIC, KII, KI2 ;
local st, st2, ineq, ineq2, i, s, anew, cnew, Bnew, sigma , sigma1 , alphas , alphas , Q, P,

```

QI, K ; with(plots, implicitplot) : print(" Start the program " );
print(" Parameters of the metal vessel and internal pressure P : ", a = aI, B = BI, 2 c = 2
· cI, d = dI, P = PI, type = typeI,  $\frac{a}{2 \cdot c} = \text{evalf}\left(\frac{aI}{2 \cdot cI}, 3\right)$ , row = k);
print(" Convert the units into meter : "); anew :=  $\frac{aI}{1000}$ ; Bnew :=  $\frac{BI}{100}$ ; cnew
:=  $\frac{cI}{1000}$ ; print(a = anew, B = Bnew, c = cnew, d = dI); P2 := PI; KI := alpha · P ·  $\sqrt{\pi \cdot a}$ ;
print(" (1) The total stress intensity factor KI " = alpha · Ppressure ·  $\sqrt{\pi \cdot a}$ ); ; alphax
:=  $\frac{1.12}{\sqrt{Q_{shape}}} \cdot \left(\frac{d_{diameter}}{2 \cdot B_{thickness}} + 1\right)$ ; print("(2) Correction factor α " =  $\frac{1.12}{\sqrt{Q_{shape}}}$ 
·  $\left(\frac{d_{diameter}}{2 \cdot B_{thickness}} + 1\right)$ ); Q :=  $\frac{\pi^2}{4} \cdot \left(\frac{3}{4} + \left(\frac{a}{2 \cdot c}\right)^2\right)^2 - \frac{7}{33} \cdot \left(\frac{\sigma}{\sigma_{ys}}\right)^2$ ; sigma :=  $\frac{P \cdot d}{2 \cdot B}$ ;
print(" (3) The hoop stress σ " =  $\frac{P_{pressure} \cdot d_{diameter}}{2 \cdot B_{thickness}}$ ); ;
print(" (4) The shape factor Q " = Qshape); printf("\n%s",
" Read the data from the temperature mechanical properties table"); printf("\n%s", " ");
print(TYPE); for i from 1 to 5 do print(s[i] = A[k][i]); s[i] := A[k][i]; od; ;
print( Material = s[1], Σys = s[2], Σts = s[3], KIc = s[4], E = s[5] ); ; printf("\n%s",
" Calculating the hoop stress sigma , the shape factor Q , correction factor alpha and the
total stress intensity factor KI "); printf("\n%s", " "); sigmaI := evalf( $\frac{PI \cdot dI}{2 \cdot Bnew}$ , 5);
print(" From (3) "); print(" (3) The hoop stress σ " =  $\frac{P_{pressure} \cdot d_{diameter}}{2 \cdot B_{thickness}}$ );
print(" σ " = sigmaI); printf("\n%s", " "); print(" From (4) ");
print(" (4) The shape factor Q " = Qshape); σys := s[2]; QI :=  $\frac{\pi^2}{4} \cdot \left(\frac{3}{4} + \left(\frac{anew}{2 \cdot cnew}\right)^2\right)^2 - \frac{7}{33} \cdot \left(\frac{\sigma I}{\sigma_{ys}}\right)^2$ ; QI := evalf(QI, 3); ; print(" Q " = QI); ;
printf("\n%s", " "); print(" From (2) "); print(" (2) Correction factor α " =  $\frac{1.12}{\sqrt{Q_{shape}}}$ 
·  $\left(\frac{d_{diameter}}{2 \cdot B_{thickness}} + 1\right)$ ); alphaI :=  $\frac{1.12}{\sqrt{QI}} \cdot \left(\frac{dI}{2 \cdot Bnew} + 1\right)$ ; alphaI := evalf(alphaI, 3);
print(" α " = alphaI); ; printf("\n%s", " "); print(" From (1) ");
print(" (1) The total stress intensity factor KI " = alpha · P ·  $\sqrt{\pi \cdot a}$ ); KI1 := alphaI · PI
·  $\sqrt{\pi \cdot anew}$ ; print(KI = KI1); KI1 := evalf(KI1, 4); print(KI = KI1); ; printf("\n%s",
" "); printf("\n%s", " 1/. Compare the total stress intensity factor KI and KIc "); ;
printf("\n%s", " "); KIC := s[4]; print(KIc = KIC);
print(" The fracture will not occur / the crack propagation is stable , as KI < KIc ");
ineq := KI1 < KIC ; print(" Analytical inequality ", ineq); KI2 := convert(taylor(KI1, t

```

```

= t0), polynom); print(" Taylor polynomial expression of KI at t = ", t0, KITaylor
= KI2); ineq2 := convert(KI2, polynom) < KIC;
print(" The analytical inequality obtained by using Taylor polynomial expression ",
ineq2); st2 := solve( {ineq2, t > 0, t < tlim}, t);
print(" Estimating time-limit based on Taylor polynomial expression of KI ", st2);
printf("\n%s", " "); st := solve( {ineq, t > 0, t < tlim}, t);
print(" Estimating time—limit based on analytical method for KI ", st); printf("\n%s",
" "); printf("\n%s",
" 2/. Determine the maximum pressure Pmax that causes the crack propagation "); ;
printf("\n%s", " "); print(" From (1) "); print( " (1) The critical stress intensity KIc "
=  $\alpha[c] \cdot P_{MAX} \cdot \sqrt{\pi \cdot a}$  ); P_MAX :=  $\frac{KIC}{\alpha l \cdot \sqrt{\pi \cdot anew}}$ ; P_MAX := evalf(P_MAX, 5); print(Pmax
= P_MAX ); ; printf("\n%s", " "); printf("\n%s",
" 3/. Consider the relation between Q and  $\alpha$  "); ; printf("\n%s", " ");
print(" From (2) "); print( " Substitute x for alpha and y for Q into (2) ");
print(alpha = alphax); x = evalf(  $\frac{1.12}{\sqrt{y}} \cdot \left( \frac{dl}{2 \cdot B_{new}} + 1 \right), 4$  ); print(%) ; printf("\n%s",
" "); ; printf("\n%s", " ");
print(" Graphical relations between t and stress intensity factor (plot1) _ Taylor series
expansion of stress intensity factor KI (plot2) " ); ; print(" End of the program " ); ; ;
printf("\n%s", " "); plot( [ KII, KI2 ], t = 0 .. tlim, y = y0 .. y1, axes = boxed, labels = [ `t`,
`KI` ], color = [ blue, green ], legend = [ KIplot1, KITaylorplot2 ], thickness = [ 3, 3 ], linestyle
= [ solid, dot ]);
end:
> pressure_KIandKIc := proc(tKcor, ymin, ymax) local pK ;
print(" The value of KIc is given from the engineering design requirement ( see table 1. ) :
KIc = ", KIC);
print(" The expresion of continous or discrete form of KI and KITaylor ");
print(" KI = ", KI(t) = KII ); print(" KITaylor = ", KITaylor(t) = KI2);
; plot( [ KII, KI2, KIC ], t = 0 .. tKcor, y = ymin .. ymax, color = [ blue,
green, red ], thickness = [ 3, 3, 3 ], legend = [ `blueKII`, `greenKITaylor`, `redKIc` ], linestyle
= [ solid, dot, dash ]);
end:
> pressure_Plotinput := proc(tcor, ymin, ymax) local pp ;
print(" The expresion of continous or discrete form of internal pressure ");
print(" Pressure function = ", f(t) = P2 ); ; plot(P2, t = 0 .. tcor, y
= ymin .. ymax, color = brown, thickness = 3, legend = `brownP` );
end:

```

Example 3.2.1

Consider a round vessel with technical data below :

one-half crack depth $a = 5\text{mm}$; thickness of vessel $B = 25\text{cm}$; one-half crack length $c = 8\text{mm}$;

internal diameter $d = 0.8\text{m}$; internal pressure $P =$

$16.13084112 \ln(t + 15.22) + 2.5 e^{\sin(0.06507451031 t + 0.6642806012) - 9600.268} \text{MPa}$; type : Ti6Al4V2 i.e
A[4][1]

a. Find the expression of stress intensity factor KI and Taylor polynomial of KI .

b. Evaluating the time-limit for applying the continous / discrete form of internal dynamic pressure .

Solution :

Run the procedure $pressure_{Estm}(\ ,\ ,\ ,\);$

```
> pressureEstm(5, 25, 8, 0.8, 16.13084112 ln(t + 15.22)
+ 2.5 esin(0.06507451031 t + 0.6642806012) - 9600.268, A[4][1], 4, 10000, 5000, 0, 80);
" Start the program "
" Parameters of the metal vessel and internal pressure P : ", a = 5, B = 25, 2 c = 16, d = 0.8, P
= 16.13084112 ln(t + 15.22) + 2.5 esin(0.06507451031 t + 0.6642806012) - 9600.268, type
= Ti6Al4V2,  $\frac{1}{2} \frac{a}{c} = 0.312$ , row = 4
" Convert the units into meter : "
 $a = \frac{1}{200}, B = \frac{1}{4}, c = \frac{1}{125}, d = 0.8$ 
" (1) The total stress intensity factor KI " =  $\alpha P_{pressure} \sqrt{\pi a}$ 
" (2) Correction factor  $\alpha$  " =  $\frac{1.12 \left( \frac{1}{2} \frac{d_{diameter}}{B_{thickness}} + 1 \right)}{\sqrt{Q_{shape}}}$ 
" (3) The hoop stress  $\sigma$  " =  $\frac{1}{2} \frac{P_{pressure} d_{diameter}}{B_{thickness}}$ 
" (4) The shape factor Q " =  $\left( \frac{1}{4} \pi^2 \left( \frac{3}{4} + \frac{1}{4} \frac{a^2}{c^2} \right)^2 - \frac{7}{33} \frac{\sigma_{ys}^2}{\sigma_{ys}^2} \right)_{shape}$ 
```

Read the data from the temperature mechanical properties table

TYPE

$s_1 = Ti6Al4V2$

$s_2 = 1007$

$s_3 = 1034$

$s_4 = 40$

$s_5 = 130$

Material = Ti6Al4V2, $\Sigma_{ys} = 1007, \Sigma_{ts} = 1034, K Ic = 40, E = 130$

Calculating the hoop stress sigma , the shape factor Q ,
correction factor alpha and the total stress intensity factor
KI

" From (3) "

$$(3) \text{ The hoop stress } \sigma = \frac{1}{2} \frac{P_{pressure} d_{diameter}}{B_{thickness}}$$

$$\sigma = 25.810 \ln(t + 15.22) + 4.0000 e^{\sin(0.065075 t + 0.66428)} - 9600.3$$

" From (4) "

$$(4) \text{ The shape factor } Q = \left(\frac{1}{4} \pi^2 \left(\frac{3}{4} + \frac{1}{4} \frac{a^2}{c^2} \right)^2 - \frac{7}{33} \frac{\sigma^2}{\sigma_{ys}^2} \right)_{shape}$$

$$Q = 1.77 - 2.09 \cdot 10^{-7} (25.8 \ln(t + 15.2) + 4.00 e^{\sin(0.0651 t + 0.664)} - 9600.)^2$$

" From (2) "

$$(2) \text{ Correction factor } \alpha = \frac{1.12 \left(\frac{1}{2} \frac{d_{diameter}}{B_{thickness}} + 1 \right)}{\sqrt{\left(\frac{1}{4} \pi^2 \left(\frac{3}{4} + \frac{1}{4} \frac{a^2}{c^2} \right)^2 - \frac{7}{33} \frac{\sigma^2}{\sigma_{ys}^2} \right)_{shape}}}$$

$$\alpha = \frac{2.91}{\sqrt{1.77 - 2.09 \cdot 10^{-7} (25.8 \ln(t + 15.2) + 4.00 e^{\sin(0.0651 t + 0.664)} - 9600.)^2}}$$

" From (1) "

$$(1) \text{ The total stress intensity factor } KI = \alpha P \sqrt{\pi a}$$

$$KI = \left(0.1455000000 (16.13084112 \ln(t + 15.22) + 2.5 e^{\sin(0.06507451031 t + 0.6642806012)} - 9600.268) \sqrt{2} \sqrt{\pi} \right) / \sqrt{1.77 - 2.09 \cdot 10^{-7} (25.8 \ln(t + 15.2) + 4.00 e^{\sin(0.0651 t + 0.664)} - 9600.)^2}$$

$$KI = \frac{0.3648 (16.13 \ln(t + 15.22) + 2.5 e^{\sin(0.06507 t + 0.6643)} - 9600.)}{\sqrt{1.77 - 2.09 \cdot 10^{-7} (25.8 \ln(t + 15.2) + 4.00 e^{\sin(0.0651 t + 0.664)} - 9600.)^2}}$$

1/. Compare the total stress intensity factor KI and KIc

$$KIc = 40$$

" The fracture will not occur / the crack propagation is stable , as KI < KIc "

" Analytical inequality "

$$\frac{0.3648 (16.13 \ln(t + 15.22) + 2.5 e^{\sin(0.06507 t + 0.6643) - 9600.})}{\sqrt{1.77 - 2.09 \cdot 10^{-7} (25.8 \ln(t + 15.2) + 4.00 e^{\sin(0.0651 t + 0.664) - 9600.})^2}} < 40$$

" Taylor polynomial expression of KI at t = ", 5000, $KITaylor = 33.34426980$
 $+ 0.0008894882453 t - 8.849970677 \cdot 10^{-8} (t - 5000)^2 + 1.175367970 \cdot 10^{-11} (t - 5000)^3$
 $- 1.756728539 \cdot 10^{-15} (t - 5000)^4 + 2.801127987 \cdot 10^{-19} (t - 5000)^5$
 " The analytical inequality obtained by using Taylor polynomial expression ",
 $0.0008894882453 t - 8.849970677 \cdot 10^{-8} (t - 5000)^2 + 1.175367970 \cdot 10^{-11} (t - 5000)^3$
 $- 1.756728539 \cdot 10^{-15} (t - 5000)^4 + 2.801127987 \cdot 10^{-19} (t - 5000)^5 < 6.65573020$
 " Estimating time-limit based on Taylor polynomial expression of KI ", $\{0. < t, t$
 $< 8155.424162\}$

" Estimating time-limit based on analytical method for KI "

2/. Determine the maximum pressure Pmax that causes the crack propagation

" From (1) "

" (1) The critical stress intensity $KIc = \alpha_c P_{Max} \sqrt{\pi a}$

$$P_{max} = 109.67 \sqrt{1.77 - 2.09 \cdot 10^{-7} (25.8 \ln(t + 15.2) + 4.00 e^{\sin(0.0651 t + 0.664) - 9600.})^2}$$

3/. Consider the relation between Q and α ;

" From (2) "

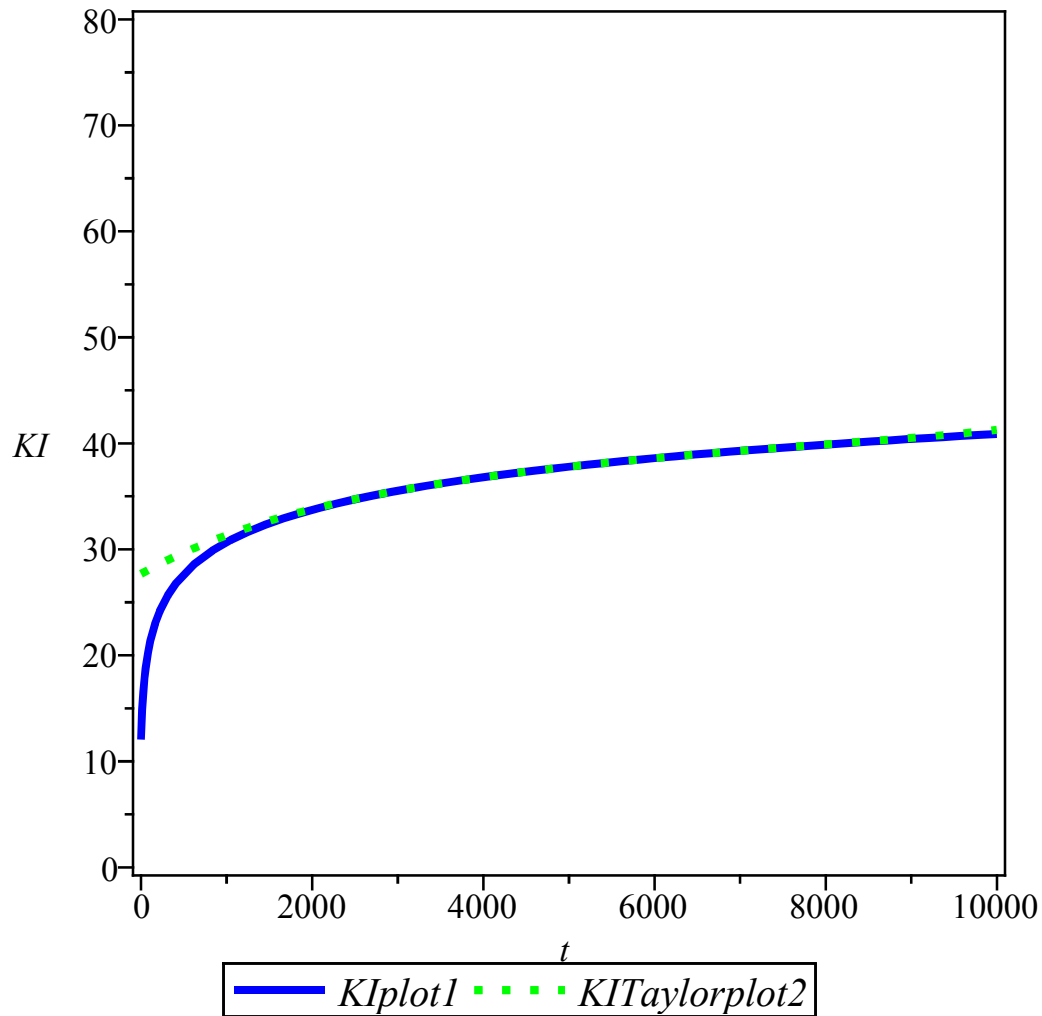
" Substitute x for α and y for Q into (2) "

$$\alpha = \frac{1.12 \left(\frac{1}{2} \frac{d_{diameter}}{B_{thickness}} + 1 \right)}{\sqrt{Q_{shape}}}$$

$$x = \frac{2.912}{\sqrt{y}}$$

" Graphical relations between t and stress intensity factor (plot1) _ Taylor series expansion of stress intensity factor KI (plot2) "

" End of the program "



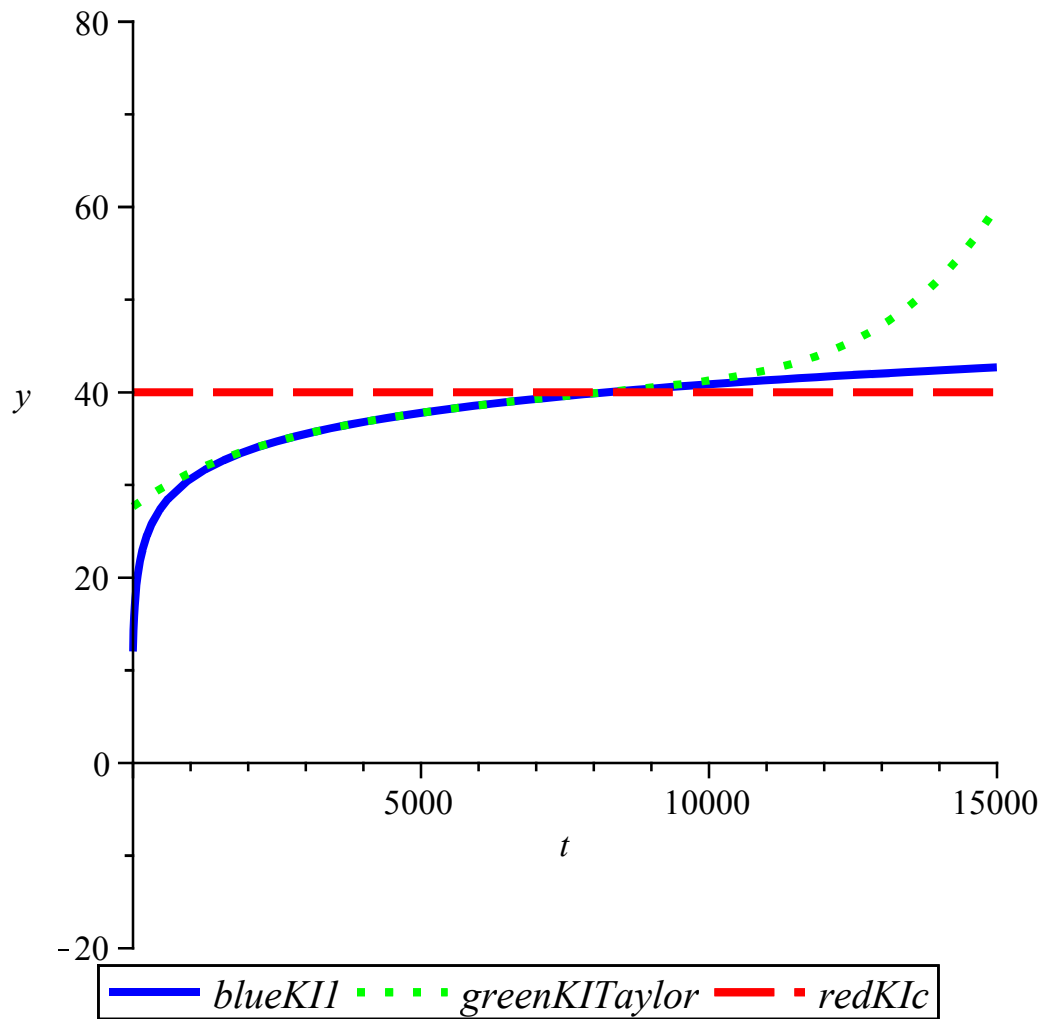
> $pressure_{KIandKIc}(15000, -20, 80)$

" The value of KIc is given from the engineering design requirement (see table 1.) : $KIc =$,
40

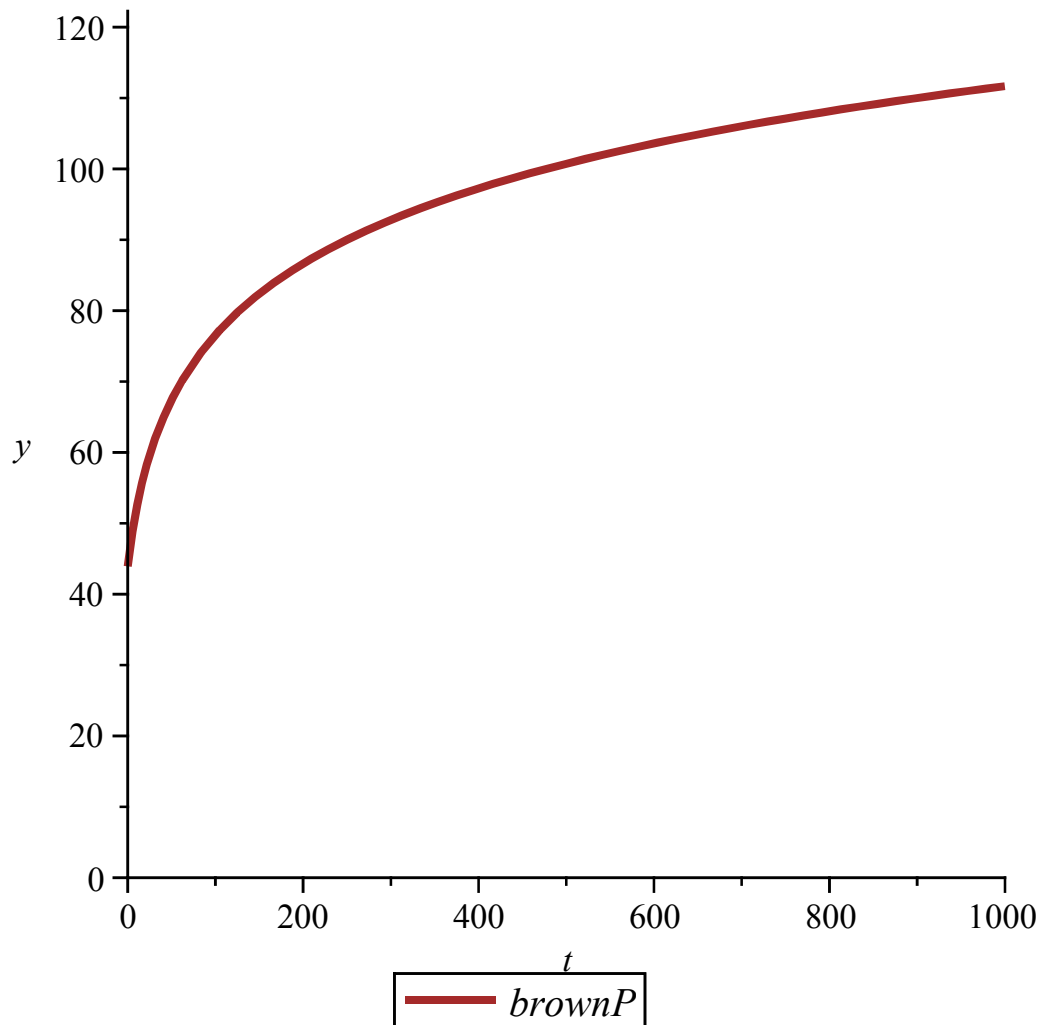
" The expresion of continous or discrete form of KI and $KITaylor$ "

$$\text{" KI = ", } KI(t) = \frac{0.3648 \left(16.13 \ln(t + 15.22) + 2.5 e^{\sin(0.06507 t + 0.6643) - 9600.} \right)}{\sqrt{1.77 - 2.09 \cdot 10^{-7} \left(25.8 \ln(t + 15.2) + 4.00 e^{\sin(0.0651 t + 0.664) - 9600.} \right)^2}}$$

$$\text{" KITaylor = ", } KITaylor(t) = 33.34426980 + 0.0008894882453 t - 8.849970677 \cdot 10^{-8} (t - 5000)^2 + 1.175367970 \cdot 10^{-11} (t - 5000)^3 - 1.756728539 \cdot 10^{-15} (t - 5000)^4 + 2.801127987 \cdot 10^{-19} (t - 5000)^5$$



```
> pressurePlotinput(1000, 00, 122.28) ;
    " The expression of continous or discrete form of internal pressure "
" Pressure function = ",  $f(t) = 16.13084112 \ln(t + 15.22)$ 
    +  $2.5 e^{\sin(0.06507451031 t + 0.6642806012)} - 9600.268$ 
```



Example 3.2.2

Consider a round vessel with technical data below :
 one-half crack depth $a = 5\text{mm}$; thickness of vessel $B = 25\text{cm}$; one-half crack length $c = 8\text{mm}$;
 internal diameter $d = 0.8\text{m}$; internal pressure P is given from the data table below ; type : 18Ni
 (250) i.e A[9][1]

18Ni (250)	<i>adjust coeff</i>	$1 / \sqrt[948]{.89 \cdot 2}$									
t	0	5	10	15	20	25	30	35	40	45	50
MPa	0	150	170	140	165	220	340	800	920	900	820

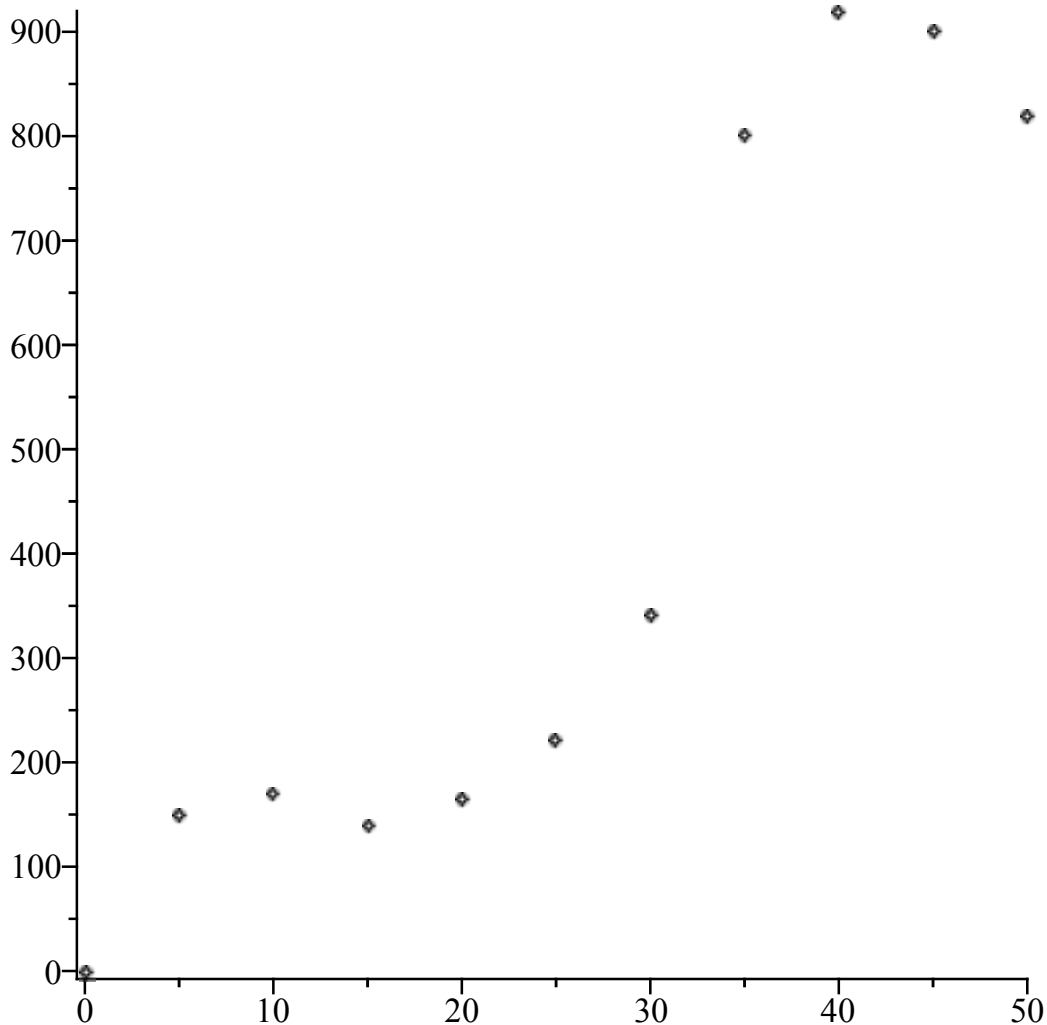
- Find the expression of stress intensity factor K_I and Taylor polynomial of K_I .
- Evaluating the time-limit for applying the discrete form of internal dynamic pressure P .

Solution :

Fitting the data points by

```
fnumPower := PolynomialInterpolation([ [0, 0.0], [5, 150], [10, 170], [15, 140], [20, 165], [25, 220], [30, 340], [35, 800], [40, 920], [45, 900], [50, 820] ], t, form = power);
```

```
> with(CurveFitting) : with(plots) : fnumPower := PolynomialInterpolation([ [0, 0.0], [5, 150], [10, 170], [15, 140], [20, 165], [25, 220], [30, 340], [35, 800], [40, 920], [45, 900], [50, 820] ], t, form = power);  
fnumPower := -6.439506133 10-10 t10 + 1.544550255 10-7 t9 - 0.00001579195757 t8  
+ 0.0008993206290 t7 - 0.03128031829 t6 + 0.6852472173 t5 - 9.393835424 t4  
+ 77.20204438 t3 - 344.1569100 t2 + 651.5571380 t  
> pointplot( { [0, 0.0], [5, 150], [10, 170], [15, 140], [20, 165], [25, 220], [30, 340], [35, 800], [40, 920], [45, 900], [50, 820] }, axes = framed);
```



Run this procedure $pressure_{Estm}(\cdot, \cdot, \cdot)$ with the approximate polynomial $fnumPower$

```
> pressureEstm(5, 25, 8, 0.8,  $\frac{fnumPower}{948.892}$ , A[9][1], 9, 80, 47, -50, 100);
```

" Start the program "

" Parameters of the metal vessel and internal pressure P : ", $a = 5$, $B = 25$, $2c = 16$, $d = 0.8$, $P = -6.786342525 \cdot 10^{-13} t^{10} + 1.627740833 \cdot 10^{-10} t^9 - 1.664252366 \cdot 10^{-8} t^8 + 9.477586793 \cdot 10^{-7} t^7 - 0.00003296509854 t^6 + 0.0007221551210 t^5 - 0.009899794098 t^4 + 0.08136020154 t^3 - 0.3626934466 t^2 + 0.6866504700 t$, type = 18 Ni250, $\frac{1}{2} \frac{a}{c} = 0.312$, row = 9

" Convert the units into meter : "

$$a = \frac{1}{200}, B = \frac{1}{4}, c = \frac{1}{125}, d = 0.8$$

" (1) The total stress intensity factor KI " = $\alpha P_{pressure} \sqrt{\pi a}$

$$\text{"(2) Correction factor } \alpha \text{"} = \frac{1.12 \left(\frac{1}{2} \frac{d_{diameter}}{B_{thickness}} + 1 \right)}{\sqrt{Q_{shape}}}$$

" (3) The hoop stress σ " = $\frac{1}{2} \frac{P_{pressure} d_{diameter}}{B_{thickness}}$

$$\text{" (4) The shape factor } Q \text{"} = \left(\frac{1}{4} \pi^2 \left(\frac{3}{4} + \frac{1}{4} \frac{a^2}{c^2} \right)^2 - \frac{7}{33} \frac{\sigma_{ys}^2}{\sigma_{ts}^2} \right)_{shape}$$

Read the data from the temperature mechanical properties table

TYPE

$$s_1 = 18 \text{ Ni250}$$

$$s_2 = 1290$$

$$s_3 = 1345$$

$$s_4 = 176$$

$$s_5 = NA$$

$$Material = 18 \text{ Ni250}, \Sigma_{ys} = 1290, \Sigma_{ts} = 1345, K_{Ic} = 176, E = NA$$

Calculating the hoop stress sigma , the shape factor Q , correction factor alpha and the total stress intensity factor KI

" From (3) "

$$\text{" (3) The hoop stress } \sigma \text{"} = \frac{1}{2} \frac{P_{pressure} d_{diameter}}{B_{thickness}}$$

$$\sigma = -1.0858 \cdot 10^{-12} t^{10} + 2.6043 \cdot 10^{-10} t^9 - 2.6629 \cdot 10^{-8} t^8 + 0.0000015164 t^7$$

$$- 0.000052745 t^6 + 0.0011554 t^5 - 0.015840 t^4 + 0.13018 t^3 - 0.58030 t^2 + 1.0986 t$$

" From (4) "

$$\text{" (4) The shape factor } Q \text{ "} = \left(\frac{1}{4} \pi^2 \left(\frac{3}{4} + \frac{1}{4} \frac{a^2}{c^2} \right)^2 - \frac{7}{33} \frac{\sigma_{ys}^2}{\sigma_{ys}^2} \right)_{shape}$$

$$\text{" } Q \text{ "} = 1.77 - 1.27 \cdot 10^{-7} (-1.09 \cdot 10^{-12} t^{10} + 2.60 \cdot 10^{-10} t^9 - 2.66 \cdot 10^{-8} t^8 + 0.00000152 t^7 - 0.0000527 t^6 + 0.00116 t^5 - 0.0158 t^4 + 0.130 t^3 - 0.580 t^2 + 1.10 t)^2$$

" From (2) "

$$\text{"(2) Correction factor } \alpha \text{ "} = \frac{1.12 \left(\frac{1}{2} \frac{d_{diameter}}{B_{thickness}} + 1 \right)}{\sqrt{\left(\frac{1}{4} \pi^2 \left(\frac{3}{4} + \frac{1}{4} \frac{a^2}{c^2} \right)^2 - \frac{7}{33} \frac{\sigma_{ys}^2}{\sigma_{ys}^2} \right)_{shape}}}$$

$$\text{" } \alpha \text{ "} = 2.91 / \left(1.77 - 1.27 \cdot 10^{-7} (-1.09 \cdot 10^{-12} t^{10} + 2.60 \cdot 10^{-10} t^9 - 2.66 \cdot 10^{-8} t^8 + 0.00000152 t^7 - 0.0000527 t^6 + 0.00116 t^5 - 0.0158 t^4 + 0.130 t^3 - 0.580 t^2 + 1.10 t)^2 \right)^{1/2}$$

" From (1) "

$$\text{" (1) The total stress intensity factor } KI \text{ "} = \alpha P \sqrt{\pi a}$$

$$KI = \left(0.1455000000 (-6.786342525 \cdot 10^{-13} t^{10} + 1.627740833 \cdot 10^{-10} t^9 - 1.664252366 \cdot 10^{-8} t^8 + 9.477586793 \cdot 10^{-7} t^7 - 0.00003296509854 t^6 + 0.0007221551210 t^5 - 0.009899794098 t^4 + 0.08136020154 t^3 - 0.3626934466 t^2 + 0.6866504700 t) \sqrt{2} \sqrt{\pi} \right) / \left(1.77 - 1.27 \cdot 10^{-7} (-1.09 \cdot 10^{-12} t^{10} + 2.60 \cdot 10^{-10} t^9 - 2.66 \cdot 10^{-8} t^8 + 0.00000152 t^7 - 0.0000527 t^6 + 0.00116 t^5 - 0.0158 t^4 + 0.130 t^3 - 0.580 t^2 + 1.10 t)^2 \right)^{1/2}$$

$$KI = \left(0.3648 (-6.786 \cdot 10^{-13} t^{10} + 1.628 \cdot 10^{-10} t^9 - 1.664 \cdot 10^{-8} t^8 + 9.478 \cdot 10^{-7} t^7 - 0.00003297 t^6 + 0.0007222 t^5 - 0.009900 t^4 + 0.08136 t^3 - 0.3627 t^2 + 0.6867 t) \right) /$$

$$\frac{(1.77 - 1.27 \cdot 10^{-7} (-1.09 \cdot 10^{-12} t^{10} + 2.60 \cdot 10^{-10} t^9 - 2.66 \cdot 10^{-8} t^8 + 0.00000152 t^7 - 0.0000527 t^6 + 0.00116 t^5 - 0.0158 t^4 + 0.130 t^3 - 0.580 t^2 + 1.10 t)^2)}{1/2}$$

1/. Compare the total stress intensity factor KI and KIc

$$KIc = 176$$

" The fracture will not occur / the crack propagation is stable , as $KI < KIc$ "

$$\begin{aligned} & \text{" Analytical inequality ", } (0.3648 (-6.786 \cdot 10^{-13} t^{10} + 1.628 \cdot 10^{-10} t^9 - 1.664 \cdot 10^{-8} t^8 \\ & + 9.478 \cdot 10^{-7} t^7 - 0.00003297 t^6 + 0.0007222 t^5 - 0.009900 t^4 + 0.08136 t^3 - 0.3627 t^2 \\ & + 0.6867 t)) / \\ & \frac{(1.77 - 1.27 \cdot 10^{-7} (-1.09 \cdot 10^{-12} t^{10} + 2.60 \cdot 10^{-10} t^9 - 2.66 \cdot 10^{-8} t^8 + 0.00000152 t^7 - 0.0000527 t^6 + 0.00116 t^5 - 0.0158 t^4 + 0.130 t^3 - 0.580 t^2 + 1.10 t)^2)}{1/2} \\ & < 176 \end{aligned}$$

$$\begin{aligned} & \text{" Taylor polynomial expression of KI at } t = 47, KItaylor = -2377.576003 \\ & + 51.69382641 t + 63.00700353 (t - 47)^2 + 88.08829310 (t - 47)^3 \\ & + 130.3049045 (t - 47)^4 + 198.6925598 (t - 47)^5 \end{aligned}$$

$$\begin{aligned} & \text{" The analytical inequality obtained by using Taylor polynomial expression ", } 51.69382641 t \\ & + 63.00700353 (t - 47)^2 + 88.08829310 (t - 47)^3 + 130.3049045 (t - 47)^4 \\ & + 198.6925598 (t - 47)^5 < 2553.576003 \end{aligned}$$

$$\text{" Estimating time-limit based on Taylor polynomial expression of KI ", } \{0. < t, t < 47.63775925\}$$

$$\begin{aligned} & \text{" Estimating time-limit based on analytical method for KI ", } \{0. < t, t < 47.52889066\}, \\ & \{71.07919106 < t, t < 72.30034108\} \end{aligned}$$

2/. Determine the maximum pressure Pmax that causes the crack propagation

" From (1) "

$$\text{" (1) The critical stress intensity } KIc = \alpha_c P_{Max} \sqrt{\pi a}$$

Pmax

$$= 482.55 (1.77 - 1.27 \cdot 10^{-7} (-1.09 \cdot 10^{-12} t^{10} + 2.60 \cdot 10^{-10} t^9 - 2.66 \cdot 10^{-8} t^8$$

$$+ 0.00000152 \, t^7 - 0.0000527 \, t^6 + 0.00116 \, t^5 - 0.0158 \, t^4 + 0.130 \, t^3 - 0.580 \, t^2 + 1.10 \, t)^2) \\ 1/2$$

3/. Consider the relation between Q and α

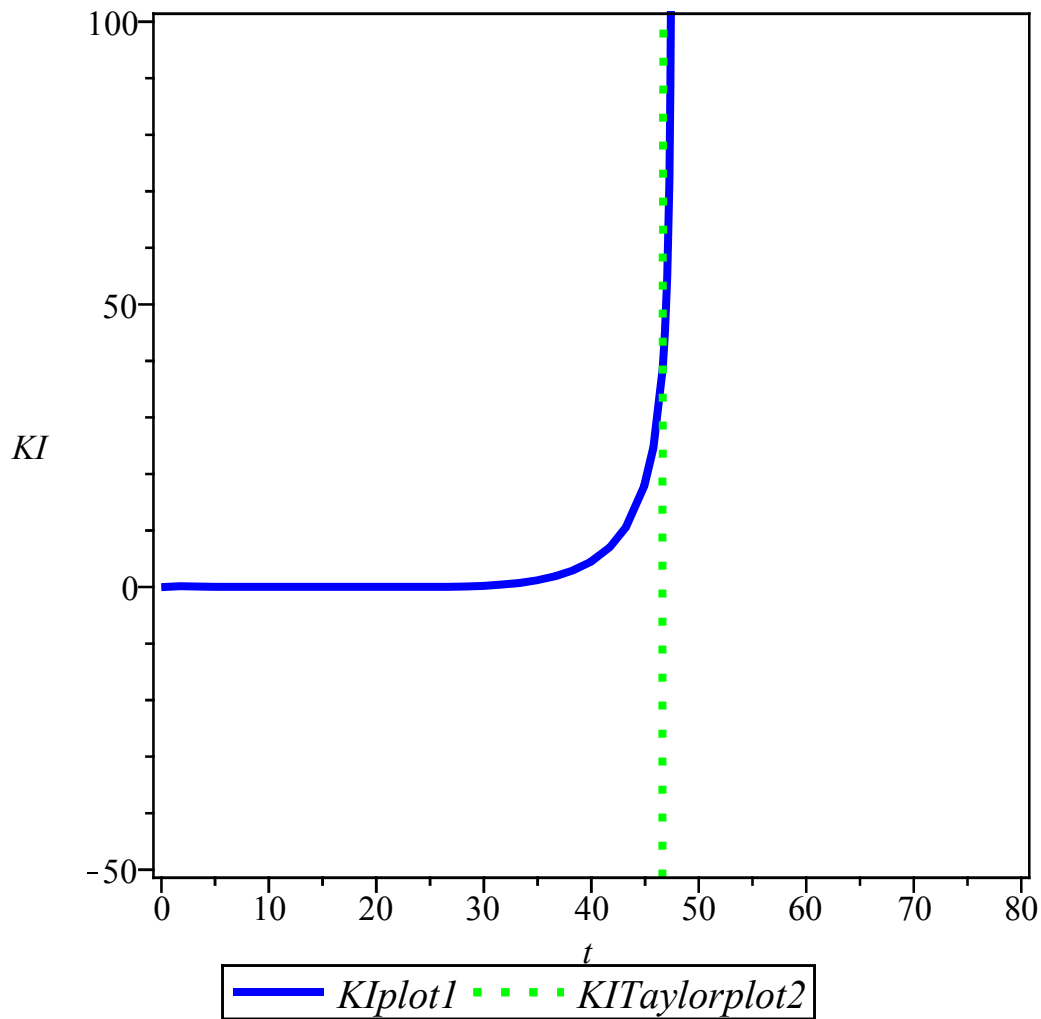
" From (2) "

" Substitute x for alpha and y for Q into (2) "

$$\alpha = \frac{1.12 \left(\frac{1}{2} \frac{d_{diameter}}{B_{thickness}} + 1 \right)}{\sqrt{Q_{shape}}} \\ x = \frac{2.912}{\sqrt{y}}$$

" Graphical relations between t and stress intensity factor (plot1) _ Taylor series expansion of stress intensity factor KI (plot2) "

" End of the program "



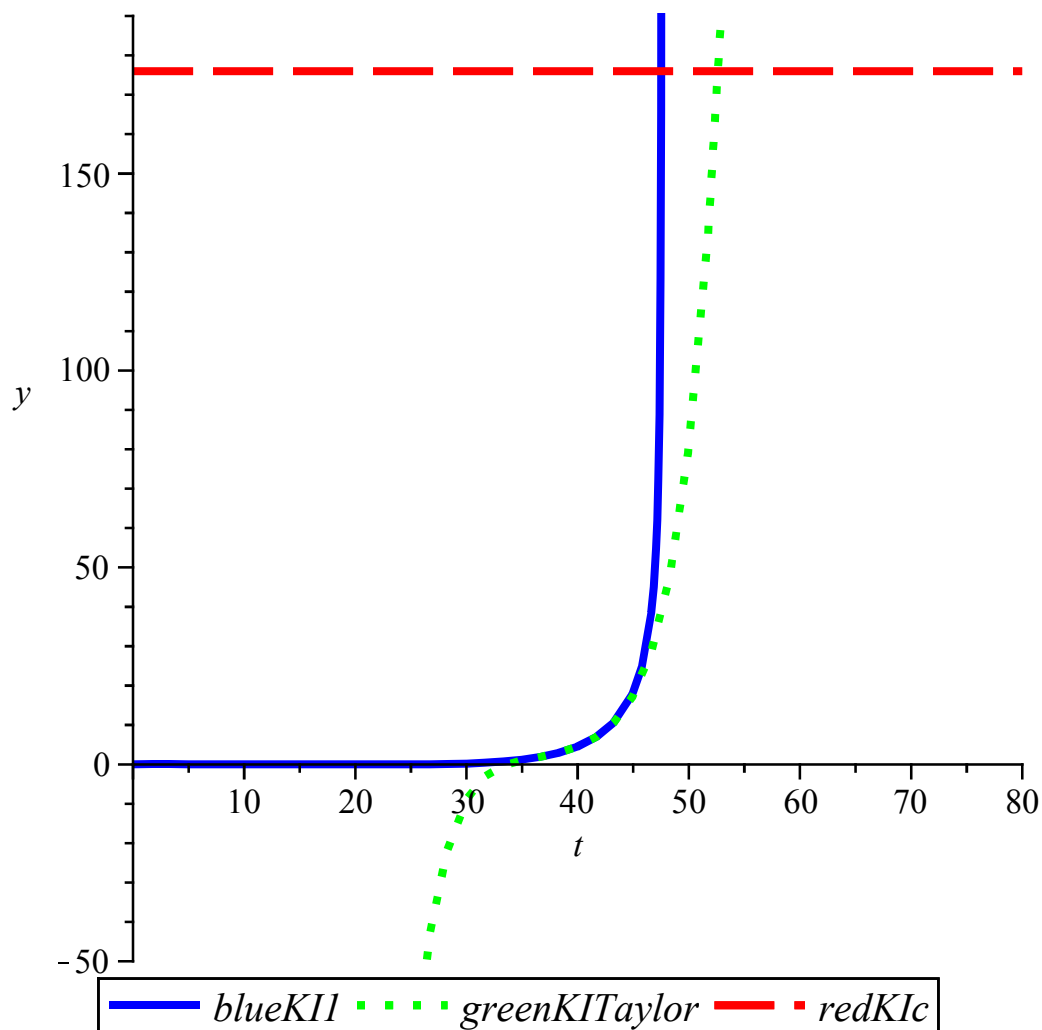
> $pressure_{KIandKIc}(80, -50, 190)$

" The value of KIc is given from the engineering design requirement (see table 1.) : $KIc =$ ",
176

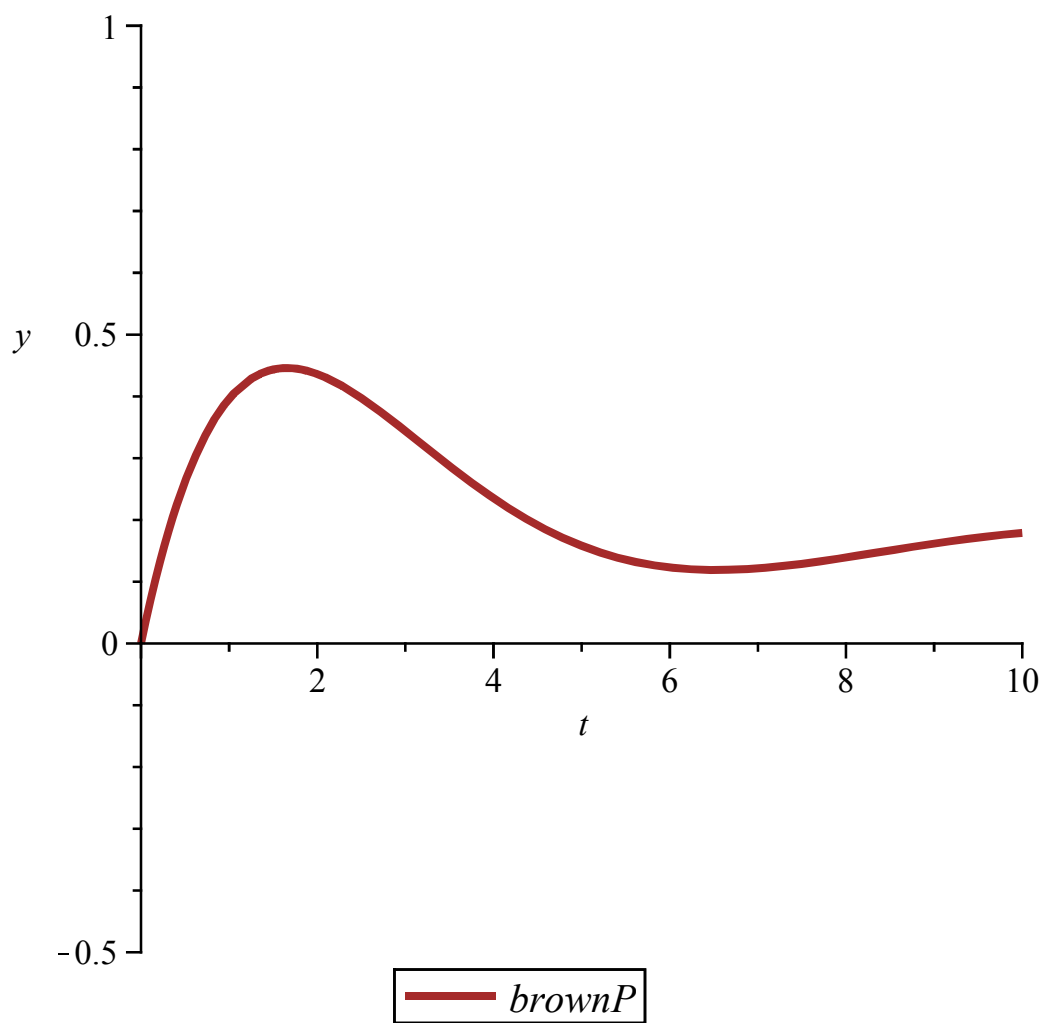
" The expresion of continous or discrete form of KI and $KITaylor$ "

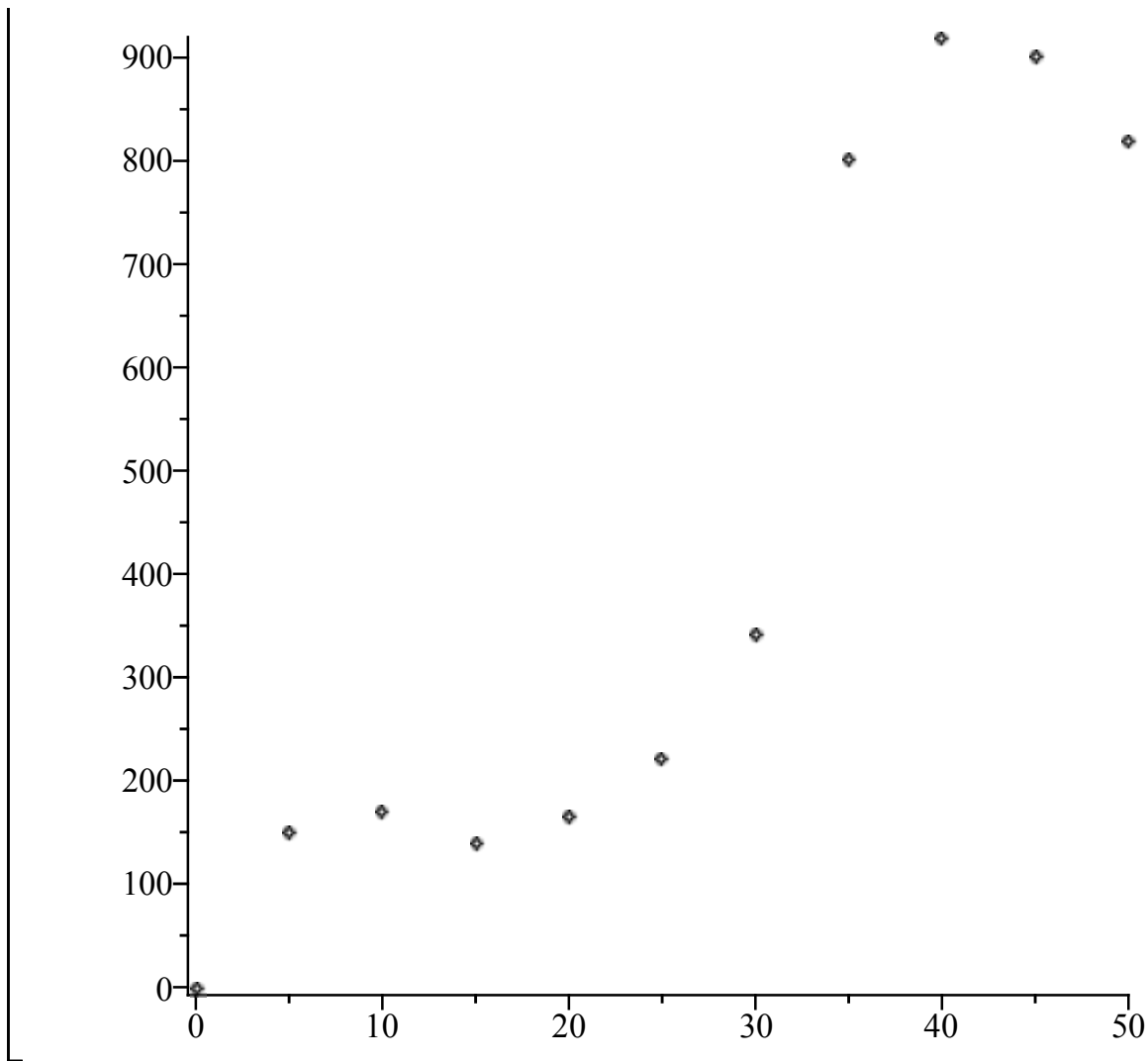
" $KI =$ ", $KI(t) = (0.3648 (-6.786 \cdot 10^{-13} t^{10} + 1.628 \cdot 10^{-10} t^9 - 1.664 \cdot 10^{-8} t^8 + 9.478 \cdot 10^{-7} t^7$
 $- 0.00003297 t^6 + 0.0007222 t^5 - 0.009900 t^4 + 0.08136 t^3 - 0.3627 t^2 + 0.6867 t)) /$
 $(1.77 - 1.27 \cdot 10^{-7} (-1.09 \cdot 10^{-12} t^{10} + 2.60 \cdot 10^{-10} t^9 - 2.66 \cdot 10^{-8} t^8$
 $+ 0.00000152 t^7 - 0.0000527 t^6 + 0.00116 t^5 - 0.0158 t^4 + 0.130 t^3 - 0.580 t^2 + 1.10 t)^2)$
 $^{1/2}$

" $KITaylor =$ ", $KITaylor(t) = -41.31589512 + 1.146168336 t + 0.1478373550 (t - 40)^2$
 $+ 0.01574740070 (t - 40)^3 + 0.001656396686 (t - 40)^4 + 0.0001763158242 (t - 40)^5$



```
> pressurePlotinput(10,-0.5,1) ; pointplot( {[0,0.0], [5,150], [10,170], [15,140], [20,165],
[25,220], [30,340], [35,800], [40,920], [45,900], [50,820] }, axes=framed);
" The expresion of continous or discrete form of internal pressure "
" Pressure function = ",  $f(t) = -6.786342525 \cdot 10^{-13} t^{10} + 1.627740833 \cdot 10^{-10} t^9$ 
 $- 1.664252366 \cdot 10^{-8} t^8 + 9.477586793 \cdot 10^{-7} t^7 - 0.00003296509854 t^6$ 
 $+ 0.0007221551210 t^5 - 0.009899794098 t^4 + 0.08136020154 t^3 - 0.3626934466 t^2$ 
 $+ 0.6866504700 t$ 
```





Notice that if we use Taylor series expansion of the approximate function for internal dynamic pressure at $t = 47$ it gives the same results of the time-limit calculated by analytical and series methods . ($0. < t, t < 47.63775925$, $0. < t, t < 47.52889066$)

Conclusion

Through the examples presented above, we find that the Maple tools has a lot of utilities in engineering computation and estimating the technical specifications . Note that in the case of applying internal dynamics pressure of discrete data we can use some softwares to find the best fitted functions , then get the expressions of the approximate functions to enter the procedures .

For the example 3.2.2 above, using the software Curve Expert , after entering the data we find the approximate expressions with parameters such as Standard Error, Correlation Coefficient ... From which we can choose the appropriate functions for the next calculation .

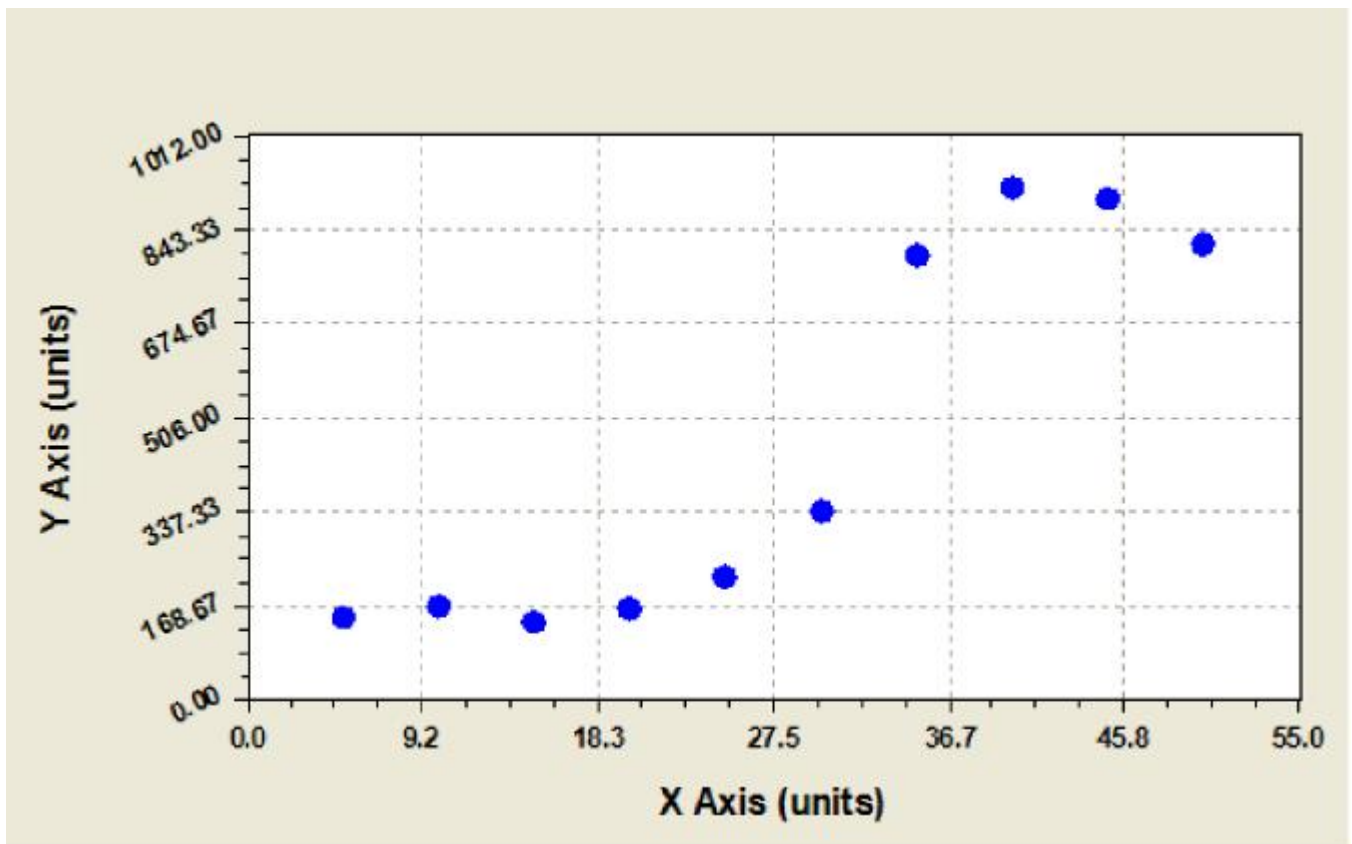


Fig 1. *Data plot* .

1. Logistic Model: $y = a / (1 + b \cdot \exp(-cx))$

Standard Error: 114.9209641

Correlation Coefficient: 0.9580550

Comments:

The fit converged to a tolerance of 1e-006 in 36 iterations. No weighting used.

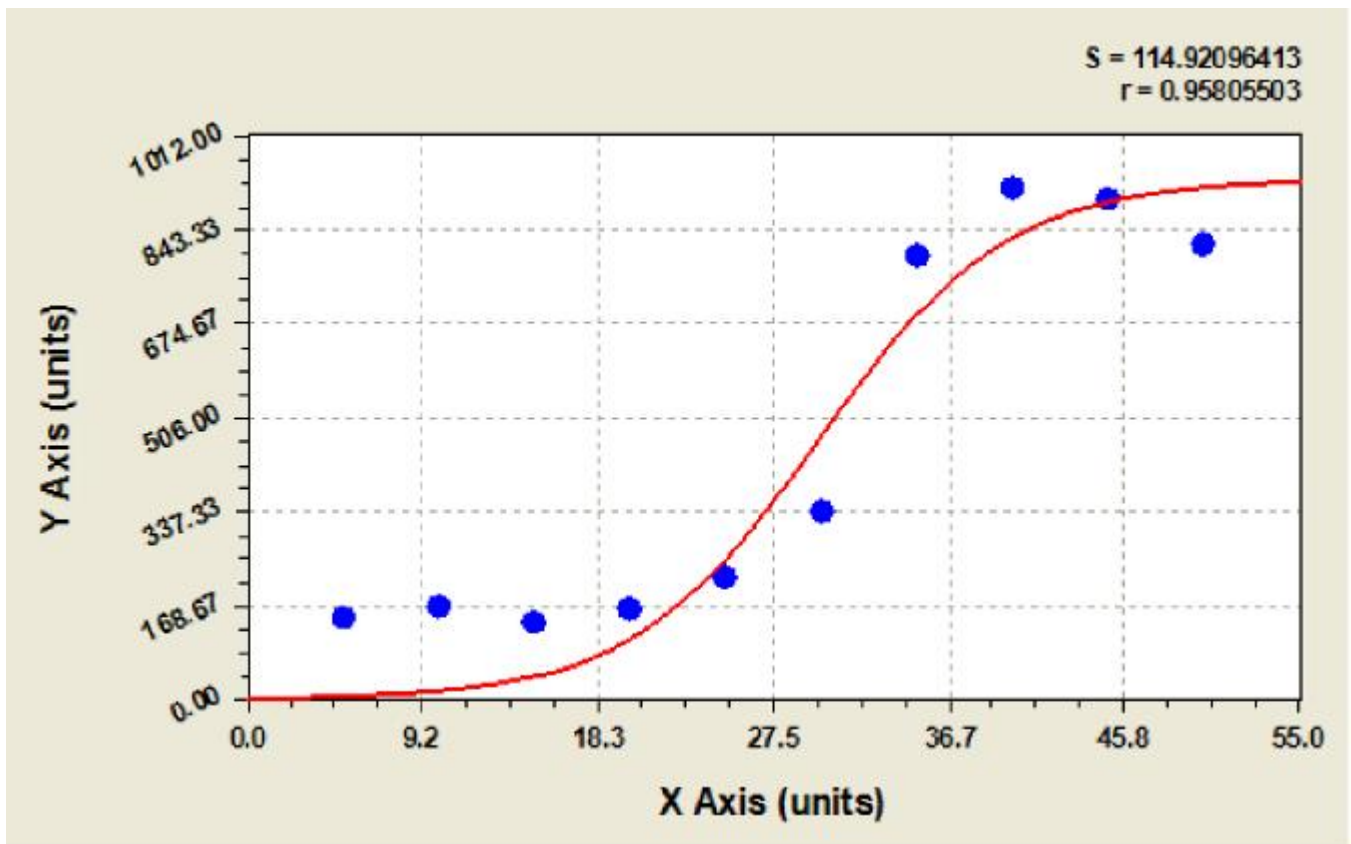


Fig 2 . Graph of the logistic fit .

2. The 4th Degree Polynomial Fit: $y=a+bx+cx^2+dx^3+ex^4$

Coefficient Data:

$a = -0.59440559$
 $b = 57.013598$
 $c = -6.1678322$
 $d = 0.23554002$
 $e = -0.0025734266$

Standard Error: 75.4632160

Correlation Coefficient: 0.9866303

Comments:

Linear regression completed successfully. No weighting used.

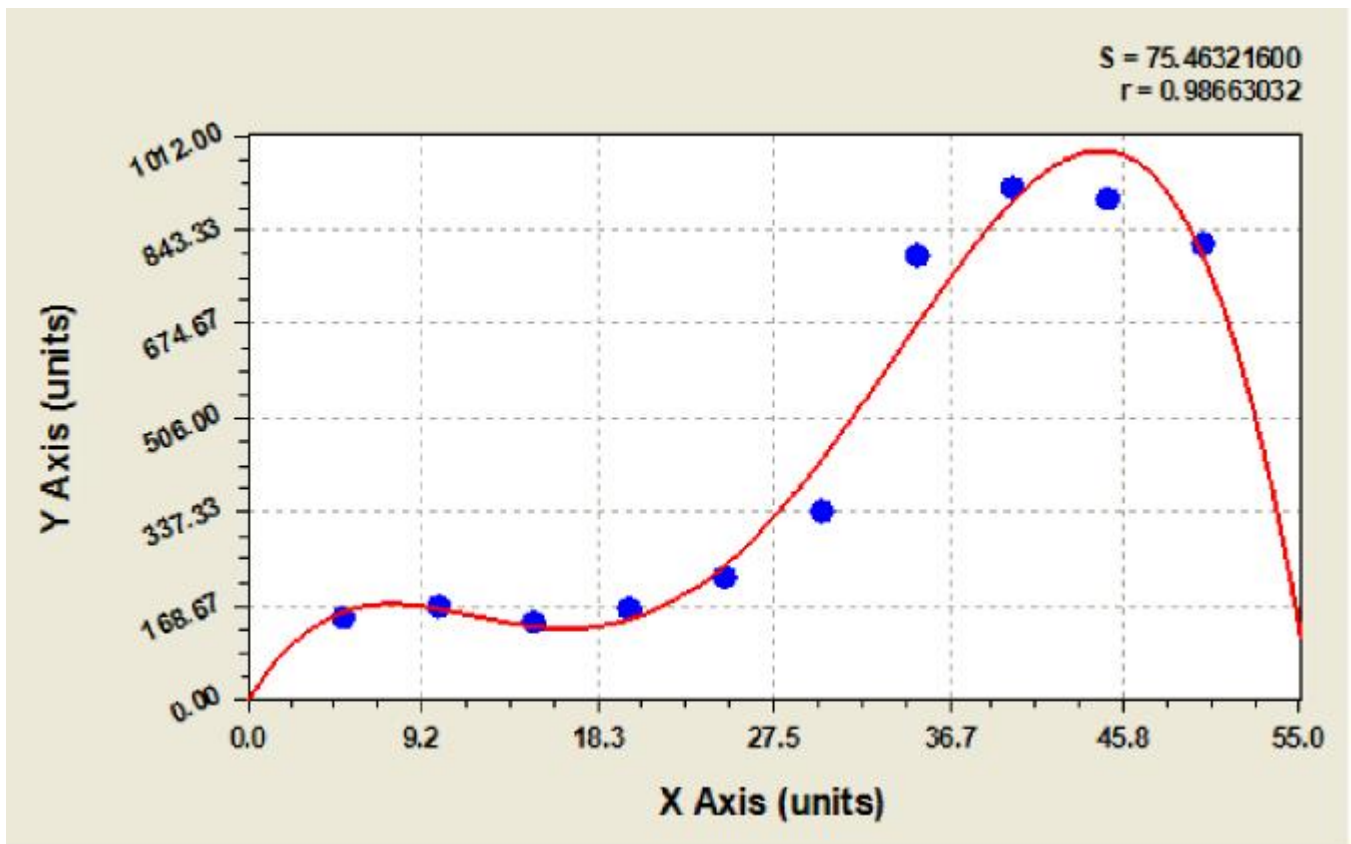


Fig 3 . Graph of the 4th Degree Polynomial Fit .

3. Lagrangian Interpolation: $y=a+bx+cx^2+dx^3+ex^4+fx^5+gx^6+hx^7+ix^8+jx^9+kx^{10}$

Coefficient Data:

$a = 0$
 $b = 651.55714$
 $c = -344.15691$
 $d = 77.202045$
 $e = -9.3938355$
 $f = 0.68524722$
 $g = -0.031280319$
 $h = 0.00089932063$
 $i = -1.5791958e-005$
 $j = 1.5445503e-007$
 $k = -6.4395062e-010$

Standard Error: 0.0000000

Correlation Coefficient: 1.0000000

Comments:

Interpolation completed successfully. Since interpolations pass through each data point, the error is 0 and the corr. coeff. is 1.

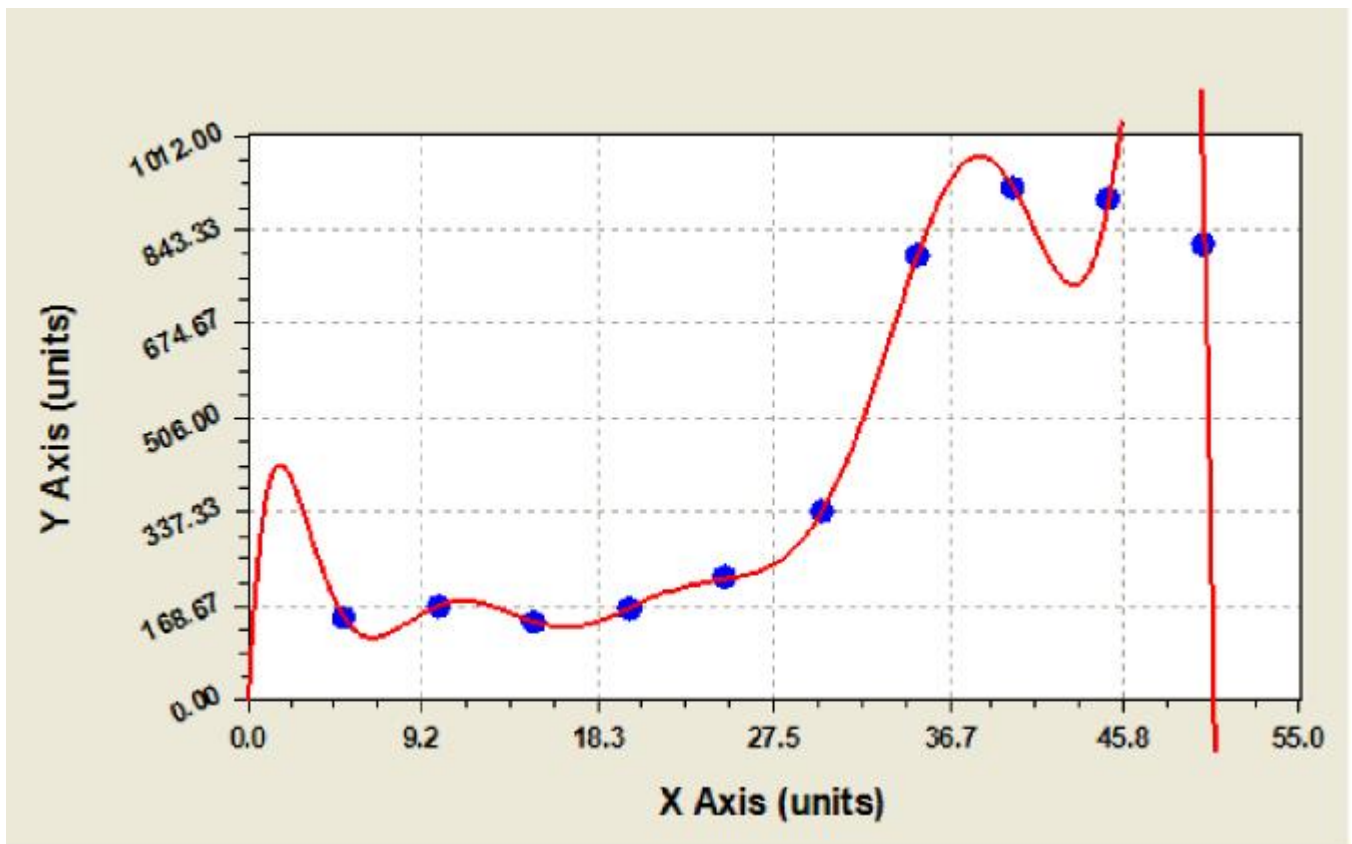


Fig 4 . Graph of the Lagrangian Interpolation fit .

REFERENCES

- [1] A.S. Koyabashi, M.Zii and L.R. Hall , *Inter. J. Fracture mechanics* ,1965.
- [2] J.M. Dahl and P.M. Novotny , *Advance Materials and Process* , 1999 .
- [3] G.R Yoder et al., ASTM STP 801 , 1983 .
- [4] R.C. Shah , ASTM STP 560 , 1971 .
- [5] J.M. Barsom and S.T. Rolfe , *Fracture and Fatigue in Structure : Application of Fracture Mechanics* , ASTM Philadelphia , PA 1999
- [6] Nestor Perez , *Fracture Mechanics* , Kluwer Academic Publisher , Boston 2004 .
- [7] Arun Shukla , *Practical Fracture Mechanics in Design* , 2nd edition , Marcel Dekker , NY 2005 .

Disclaimer: While every effort has been made to validate the solutions in this worksheet, the author is

not responsible for any errors contained and are not liable for any damages resulting from the use of this material.

Legal Notice: Maplesoft, a division of Waterloo Maple Inc. 2009. Maplesoft and Maple are trademarks of Waterloo Maple Inc. The copyright for this application is owned by the author . Neither Maplesoft nor the authors are responsible for any errors contained within and are not liable for any damages resulting from the use of this material. This application is intended for non-commercial, non-profit use only. Contact the authors for permission if you wish to use this application in for-profit activities.

Thank you for evaluating this Maple application sample

www.maplesoft.com